

UFOP, Departamento de Engenharia Civil 6th May 2014



# Nonlinear Multi-Scale Finite Element Modeling of the Progressive Collapse of Reinforced Concrete Structures

### **Lecture 3: Computational plasticity**

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Inspired and adapted from the 'Nonlinear Modeling of Structures' course of Prof. Thierry J. Massart at the ULB



## Sources of nonlinearity 2

### Non proportionnality between applied forces and displacements







# Micromechanical origins of plasticity 3



Permanent deformation induced by glide on crystallographic planes



## Micromechanical origins of plasticity 4





<u>Perfect plasticity:</u> glide of cristallographic planes under constant stress Irreversibility manifests itself through **permanent strains** 

Reversible stress states (without permanent strains) are limited by the stress level  $\sigma_y$  (states  $\sigma > \sigma_y$  are impossible with permanent strains unchanged)



Reversibility of a state change depends the stress



# Hardening plastic model (macroscale) 5

- Glide of cristallographic planes impeded by dislocations
- Increase of  $\sigma$  needed to produce further plastic strains



B - C : extension of the set of admissible stress states without further permanent strain increase





## Atomic scale model of plastic indentation 6







### Interaction of dislocations + Contact evolution = Increasing stress necessary to induce further penetration



## Course planning 7

Phenomenological material model of constituents of RC
Uniaxial and multiaxial plastic constraints
Constitutive law in the structural solution procedure
Implicit nature of the problem at Gauss points
Return Mapping (uniaxial case)
Loops in solution procedure with plasticity
Beam-column plastic frame example
Elastic vs. plastic modeling





## From material to structural behavior 8







## Concrete material behavior 9



[The International Federation for Structural Concrete (fib)]

- E strain rate dependent

$$\frac{E_c}{E_{c,st}} = \left(\frac{\dot{\epsilon}}{\dot{\epsilon}_{st}}\right)^{0.05}$$

- Plasticity strain rate dependent

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$$\frac{f_c}{f_{c,st}} = \begin{cases} \left(\frac{\dot{\epsilon}}{\dot{\epsilon}_{st}}\right)^{1.026\alpha_s} & \text{for } \dot{\epsilon} \le 30 \text{ s}^{-1} \\ \gamma_s \left(\frac{\dot{\epsilon}}{\dot{\epsilon}_{st}}\right)^{1/3} & \text{for } \dot{\epsilon} > 30 \text{ s}^{-1} \end{cases}$$

Perzyna viscoplastic model

$$\dot{\epsilon}^{vp} = \frac{1}{\eta} \left\langle \frac{f}{\bar{\sigma}_0} \right\rangle^N \frac{\mathrm{d}f}{\mathrm{d}\sigma}$$





# 1D Constitutive models of constituents 10

1D laws fitted to experimental data



- Plasticity strain rate dependent
- Ultimate cutoff strain



Concrete

- E strain rate dependent
- Plasticity strain rate dependent
- No resistance to traction
- Ultimate cutoff strain





## Uniaxial plasticity 11



#### Hardening parameter $\kappa$

History parameter representing the cumulated effect of plastic dissipation

Current yield strength  $\sigma_{y}$ 

Stress that has to be exceeded to induce further plastic straining

### Hardening

If the plastic strain increases the admissible domain expands (if  $\sigma_y(\kappa)$  is an increasing function)







Hardening behaviorStrain rate independent

## Uniaxial plasticity 12

Strain partition  $\varepsilon = \varepsilon^e + \varepsilon^p$ 

Plasticity criterion in stress space

$$\begin{split} f^{p} &= \sigma - \sigma_{y}\left(\kappa\right) = 0 \rightarrow \text{Increase of } \varepsilon^{p} \\ f^{p} &= \sigma - \sigma_{y}\left(\kappa\right) < 0 \rightarrow \text{Elastic behaviour} \\ f^{p} &= \sigma - \sigma_{y}\left(\kappa\right) > 0 \rightarrow \text{Non admissible states} \end{split}$$

Constitutive relation

$$\sigma = E\varepsilon^e = E\left(\varepsilon - \varepsilon^p\right)$$

#### **Consistency condition**

The point representing the stress state in the stress space has to remain on the reversible domain when plastic strains are increasing







Multiaxial plasticity – yield surface 13

Strain partition  $\varepsilon = \varepsilon^e + \varepsilon^p$ Plasticity criterion in stress space (isotropic) principal stresses Tresca  $f^p = \sigma_{eq} (\sigma_1, \sigma_2, \sigma_3) - \sigma_y (\kappa) = 0 \rightarrow \text{Increase of } \varepsilon^p$  $f^{p} = \sigma_{eq} (\sigma_{1}, \sigma_{2}, \sigma_{3}) - \sigma_{y} (\kappa) < 0 \rightarrow \text{Elastic behaviour}$  $f^p = \sigma_{eq} (\sigma_1, \sigma_2, \sigma_3) - \sigma_y (\kappa) > 0 \rightarrow \text{Non admissible states}$ 



Constitutive relation  $\{\sigma\} = [H] \{\varepsilon^e\} = [H] (\{\varepsilon\} - \{\varepsilon^p\})$ 

**Consistency condition** 

'Direction' of plastic strains

 $\{d\varepsilon^p\} = d\lambda \left\{\frac{\partial g^p}{\partial \sigma}\right\} \quad g^p = f^p \text{ most often chosen for metals}$ 



## Structure of a NL FE code 14

Define a set of successive loading states  $F_{ext,n}$ Loop on the loading states (steps or increments) Formulate the problem for the step  $n \rightarrow n+1$ Find  $q_{n+1}$  such that  $F_{int}(q_{n+1}) - F_{ext,n+1} = 0$ With as first approximation  $q_{n+1}^{(0)} = q_n$ Iterate until a precision threshold is reached with  $t+\Delta t \{f_{int}\}^{(i)} = \int_{V_e} [B] t+\Delta t \{\sigma\}^{(i)} dV_e$   $q_{n+1}^{(k+1)} = q_{n+1}^{(k)} - \left[\left(\frac{\partial F_{int}}{\partial q}\right)\right]_{q_{n+1}^{(k)}}^{-1} \left(F_{ext,n+1} - F_{int,n}^{(k)}\right)$   $q_{n+1}^{(k)} = q_{n+1}^{(k)} - \left[\left(\frac{\partial F_{int}}{\partial q}\right)\right]_{q_{n+1}^{(k)}}^{-1} \left(F_{ext,n+1} - F_{int,n}^{(k)}\right)$ End of increment





## Constitutive law in solution procedure 15

Structural tangent stiffness at iteration (i)  

$$\underbrace{\left[\frac{\partial t^{+\Delta t} \{f_{int}\}^{(i-1)}}{\partial q}\right]}_{[K_t(t^{+\Delta t}\{q\}^{(i-1)})]} t^{+\Delta t} \{\delta q\}^{(i)} \simeq t^{+\Delta t} \{f_{ext}\} - t^{+\Delta t} \{f_{int}\}^{(i-1)}$$

$$t^{+\Delta t} [K_t]^{(i-1)} = \sum_{e} \left( \int_{V_e} [B]^T t^{+\Delta t} [\mathbf{L}]^{(i-1)} [B] \, \mathrm{dV_e} \right)$$
with  $\left\{ \Delta \sigma \} = \underbrace{t}_{e} \left[ \frac{\partial \sigma}{\partial \varepsilon} \right] \{\Delta \varepsilon \}$ 
Internal force computation at iteration (i)
$$t^{+\Delta t} \{q\}^{(i)} \rightarrow t^{+\Delta t} \{\varepsilon\}^{(i)} \rightarrow t^{+\Delta t} \{\sigma\}^{(i)}$$

$$\rightarrow t^{+\Delta t} \{f_{int}\}^{(i)} = \int_{v} [B] t^{+\Delta t} \{\sigma\}^{(i)} dv$$





## Computation of internal forces 16

$$\begin{split} {}^{t+\Delta t} \left\{ f_{int} \right\}^{(i)} &= \int_{V_e} \left[ B \right] \, {}^{t+\Delta t} \left\{ \sigma \right\}^{(i)} \, \mathrm{dV_e} \\ \\ \text{Stress update at iteration (i) at each Gauss point } \sigma(\epsilon) \\ \\ \text{Known values} \quad {}^t \left\{ \sigma \right\}, \, {}^t \left\{ \varepsilon \right\}, \, {}^t \left\{ \varepsilon^p \right\}, \, {}^t \kappa \, \, (\text{last converged configuration}) \\ \\ \text{Strain update} \quad {}^{t+\Delta t} \left\{ \Delta \varepsilon \right\}^{(i)} &= \left[ B \right] \, {}^{t+\Delta t} \left\{ \Delta q \right\}^{(i)} \\ \\ \\ \text{To be determined} \quad {}^{t+\Delta t} \left\{ \sigma \right\}, \, {}^{t+\Delta t} \left\{ \varepsilon^p \right\}, \, {}^{t+\Delta t} \kappa \end{split}$$

Implicit nature of the problem in each Gauss point

The strain update is known  ${}^{t+\Delta t}\epsilon = {}^t\epsilon + {}^{t+\Delta t}\Delta\epsilon$ 

But the reversibility criterion is expressed in the stress space

$$\sigma_{eq}\left(^{t+\Delta t}\left\{\sigma\right\}^{(i)}\right) - \sigma_y\left(^{t+\Delta t}\kappa^{(i)}\right) = 0$$

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where  ${}^{t+\Delta t}\left\{\sigma
ight\}^{(i)}$  and  ${}^{t+\Delta t}\kappa^{(i)}$  are unknown



## Implicit problem at Gauss points 17

The strain update is known 
$$^{t+\Delta t}\epsilon = ^{t}\epsilon + {}^{t+\Delta t}\Delta \epsilon$$
  
 $^{t+\Delta t}\Delta \epsilon = {}^{t+\Delta t}\Delta \epsilon^{e(i)} + {}^{t+\Delta t}\Delta \epsilon^{p(i)}$   
 $^{t+\Delta t}\Delta \sigma^{(i)} = E\left({}^{t+\Delta t}\Delta \epsilon - {}^{t+\Delta t}\Delta \epsilon^{p(i)}\right)$   
 $^{t+\Delta t}\sigma^{(i)} = {}^{t}\sigma + {}^{t+\Delta t}\Delta \sigma^{(i)}$   
 $^{t+\Delta t}\kappa^{(i)} = {}^{t}\kappa + {}^{t+\Delta t}\Delta \kappa^{(i)}$   
if  $f_p\left({}^{t+\Delta t}\sigma^{(i)}, {}^{t+\Delta t}\kappa^{(i)}\right) > 0$ 

- decrease  ${}^{t+\Delta t}\sigma^{(i)}$  by increasing plastic strain  ${}^{t+\Delta t}\Delta\kappa^{(i)}$ 

simultaneously!

- increase in  ${}^{t+\Delta t}\Delta\kappa^{(i)}$  inflates  $f_p\left({}^{t+\Delta t}\sigma^{(i)}, {}^{t+\Delta t}\kappa^{(i)}\right)$ 





# Solution - Return Mapping (1D case) 18

#### At each Gauss point

 $\sigma^{(k)} = \sigma_c + \mathbf{E} \left( \Delta \epsilon - \Delta \epsilon_p^{(k)} \right)$ 

 $\sigma_{tr} = \sigma_c + E \Delta \epsilon$  trial stress - elastic increment assumption

if 
$$f_p\left(\sigma_{tr}, {}^t\kappa\right) > 0$$
  
 $\sigma^{(k)} = \sigma_{tr} - E\Delta\epsilon_p^{(k)}$   
 $\frac{\sigma^{(k)} - \sigma_{tr}}{E} + \Delta\epsilon_p^{(k)} = 0$ 

solve using Newton-Raphson

$$\begin{cases} \frac{\sigma^{(k)} - \sigma_{tr}}{\mathbf{E}} + \Delta \epsilon_p^{(k)} = 0\\ f(\sigma^{(k)}, \kappa^{(k)}) = 0 \end{cases}$$

$$\begin{cases} \sigma^{(k+1)} \\ \kappa^{(k+1)} \end{cases} = \begin{cases} \sigma^{(k)} \\ \kappa^{(k)} \end{cases} - [\mathbf{J}_p(\sigma^{(k)}, \kappa^{(k)})]^{-1} \begin{cases} R^{(k)} \\ R^{(k)} \end{cases}$$
 with  $\mathbf{J}_p(\sigma^{(k)}, \kappa^{(k)}) = \begin{bmatrix} \frac{\partial R_{\epsilon}}{\partial \sigma} & \frac{\partial R_{\epsilon}}{\partial \kappa} \\ \frac{\partial R_f}{\partial \sigma} & \frac{\partial R_f}{\partial \kappa} \end{bmatrix}$ 





## Computation of internal forces - summary 19

 ${}^{t+\Delta t} \{f_{int}\}^{(i)} = \int_{V_e} [B] {}^{t+\Delta t} \{\sigma\}^{(i)} dV_e$ Determine  ${}^{t+\Delta t} \{\sigma\}^{(i)}$ , knowing that the yield surface evolves A system of equations needs to be solved This system is nonlinear if  $\sigma_y(\kappa)$  is a nonlinear function

'Local' (at each Gauss point) problem solved by Newton Raphson Initialise using the elastic predictor if outside of the yield surface 'bring back' the point on the yield

if outside of the yield surface 'bring back' the point on the yield surface by solving the equations

Material tangent stiffness = by-product of this stress update

$$\{\Delta\sigma\} = \underbrace{t \left[\frac{\partial\sigma}{\partial\varepsilon}\right]}_{t[\mathbf{L}]} \{\Delta\varepsilon\}$$





Loops in the solution procedure for plasticity 20

Loop on loading (for) Initialise residual <sup>2</sup>Loop on iterations (while residual > tolerance) 2 Assembly of stiffness (Loop on elements) 3  $^{t+\Delta t} [K_t]^{(i-1)} = \sum \left( \int_V [B]^T t + \Delta t [\mathbf{L}]^{(i-1)} [B] dV_e \right)$ **Computation reaction forces** Compute internal forces (loop on elements) 4  $^{t+\Delta t} \{f_{int}\}^{(i)} = \int_{V} [B] \ ^{t+\Delta t} \{\sigma\}^{(i)} \ \mathrm{dV}_{\mathrm{e}}$ Compute stresses at Gauss points Local Newton Raphson at Gauss points (bring back point on yield) 5 Evaluate new residual End of iteration loop End of loop on loading





### Structure of a NL FE code 21 Graphical interpretation





## Plasticity - summary 22

- Permanent strain

 $\varepsilon = \varepsilon^e + \varepsilon^p$ 

- Constitutive law (in 1D)

$$\sigma = E\varepsilon^e = E\left(\varepsilon - \varepsilon^p\right)$$

- Stress treshold

$$f^{p} = \sigma_{eq} \left( \sigma_{1}, \sigma_{2}, \sigma_{3} \right) - \sigma_{y} \left( \kappa \right) = 0$$

- History variable

Cumulated plastic strain

- Hardening/evolution law  $\sigma_{y}\left(\kappa
ight)$ 





- Implicit nature; RM at Gauss pts

































# Plastic behavior of beam-column frames

### Elastic beam elements linked by 'hinges'

Rigid behaviour in elastic phase

Plastic behaviour after reaching plastic moment limit

### Plasticity criteria in M-N variables

### **Evolution law**

Nodes with same

positions







Plastification in (M-N) -Symmetric cross-section with identical behaviour in tension and in compression Plastification by M only (simplification)



## Comparison with a nonlinear approach Complete non linear computation on a 'simple' structure



Simulate (in simplified way) the cause of the initial failure The failing element is present during the whole computation Large displacements (to model the instability of the initially failed element)





# Comparison with a nonlinear approach

Complete computation

Elastic successive steps



The failure schemes are completely different





## Comparison with a nonlinear approach Initially failing element replaced by its forces which are decreased progressively



'Incomplete' non linear computation



Replace failing element by its M,N,T Decrease these M,N,T progressively

The failure modes are much more in agreement





### Recommended literature 32

M.A. **Crisfield**, Non-linear Finite Element Analysis of Solids and Structures **Volume 1**: ESSENTIALS. John Wiley & Sons Ltd. Bafins Lane, Chichester West Sussex PO19 IUD, England, 1991.

B.S. Iribarren, P. Berke, Ph. Bouillard, J. Vantomme, T.J. Massart, Investigation of the influence of design and material parameters in the progressive collapse analysis of RC structures, *Engineering Structures*, Vol. 33, page 2805-2820, 2011.

